

Oscillatory Gap Damping

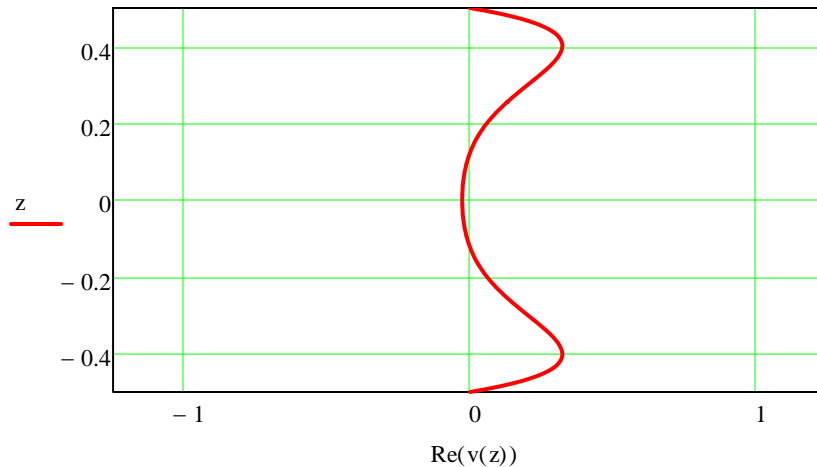
Find the damping due to the linear motion of a viscous gas in a gap with an oscillating size: 1) Find the motion in a gap due to an oscillating external force; 2) Recast the solution in terms of an effective lateral bulk drag term; 3) Solve the Fourier-periodic solution driven by an equivalent uniform mass source.

1) Here's the solution for oscillatory flow in a gap driven by an externally applied force: $\frac{lg}{2} < z < \frac{lg}{2}$

$$\frac{d}{dt}v = D \cdot \frac{d^2}{dz^2}v + f_0 \cdot e^{-i \cdot \omega \cdot t} \quad D = \frac{\gamma}{\rho} \quad f_0 \quad \text{external applied force per mass}$$

$$v(z, t) = v_0 \cdot \left(1 - \frac{\cosh(k \cdot z)}{\cosh\left(k \cdot \frac{g}{2}\right)} \right) \cdot e^{-i \cdot \omega \cdot t} \quad k = \frac{1 - i}{\delta} \quad \delta = \sqrt{\frac{2 \cdot \gamma}{\omega \cdot \rho}} \quad v_0 = \frac{f_0}{-i \cdot \omega}$$

$$\delta := 0.125 \quad \varphi := 2 \cdot \pi \cdot \text{FRAME} \quad z := -0.5, -0.495 \dots 0.5 \quad v(z) := i \cdot \left[1 - \frac{\cosh\left[(1 - i) \cdot \frac{z}{\delta}\right]}{\cosh\left(\frac{1 - i}{2 \cdot \delta}\right)} \right] \cdot \exp(-i \cdot \varphi)$$



For large drag (δ), the velocity is in phase with the force. Small drag has the velocity lagging the force by $\pi/2$ phase.

2) Now get the average velocity:

$$v_{av} = \frac{2}{g} \int_0^{\frac{g}{2}} v(z) dz = \frac{2}{g} \int_0^{\frac{g}{2}} v_0 \left(1 - \frac{\cosh(k \cdot z)}{\cosh\left(k \cdot \frac{g}{2}\right)} \right) dz = v_0 \left(1 - \frac{2}{k \cdot g} \cdot \tanh\left(k \cdot \frac{g}{2}\right) \right)$$

Describe by a bulk drag coefficient b
(drag force per mass)

$$\frac{dv_{av}}{dt} = -b \cdot v_{av} + f_0 \quad v_{av} = \frac{f_0}{b - i \cdot \omega}$$

Choose b to force the two solutions agree:

$$\frac{f_0}{b - i \cdot \omega} = \frac{f_0}{-i \cdot \omega} \left(1 - \frac{2}{k \cdot g} \cdot \tanh\left(k \cdot \frac{g}{2}\right) \right)$$

$$b(\omega) = i \cdot \omega \cdot \left(1 - \frac{1}{1 - \frac{2 \cdot \Delta}{1 - i} \cdot \tanh\left(\frac{1 - i}{2 \cdot \Delta}\right)} \right) \quad \Delta = \frac{\delta}{g} = \sqrt{\frac{2 \cdot \gamma}{\omega \cdot \rho \cdot g^2}}$$

$$b(\Delta) = \frac{2 \cdot \gamma}{\rho \cdot g^2} \cdot G(\Delta) \quad G(\Delta) := \frac{i}{\Delta^2} \cdot \left(\frac{1}{1 - \frac{1 - i}{2 \cdot \Delta} \cdot \coth\left(\frac{1 - i}{2 \cdot \Delta}\right)} \right)$$

$$\text{small } \Delta \dots \quad G(\Delta) = \frac{1 - i}{\Delta} \quad b = \sqrt{\frac{2 \cdot \gamma \cdot \omega}{\rho \cdot g^2}} \cdot (1 - i) = \frac{\delta}{g} \cdot \omega \cdot (1 - i)$$

The imaginary part reduces the average equivalent flow by removing δ from the undamped flow.

$$\text{large } \Delta \dots \quad G(\Delta) = 6 \quad b = \frac{12 \cdot \gamma}{\rho \cdot g^2} \quad \text{large drag}$$

Wave Propagating into the Gap

Now consider a 1D compressible wave propagation into in a narrow gap...

$$e^{i \cdot (k \cdot x - \omega \cdot t)}$$

fluid acceleration

$$\frac{\delta v}{\delta t} = -\frac{1}{\rho} \cdot \frac{\delta P}{\delta x} - b \cdot v$$

conservation of mass

$$\frac{1}{c^2} \cdot \frac{\delta P}{\delta t} = -\rho \cdot \frac{\delta v}{\delta x}$$

$$-i \cdot \omega \cdot v = -\frac{i \cdot k}{\rho} \cdot P - b \cdot v \quad \frac{-i \cdot \omega}{c^2} \cdot P = -\rho \cdot i \cdot k \cdot v$$

$$\begin{bmatrix} i \cdot k & (b - i \cdot \omega) \cdot \rho \\ -i \cdot \omega & i \cdot k \cdot \rho \cdot c^2 \end{bmatrix} \cdot \begin{pmatrix} P \\ v \end{pmatrix} = 0$$

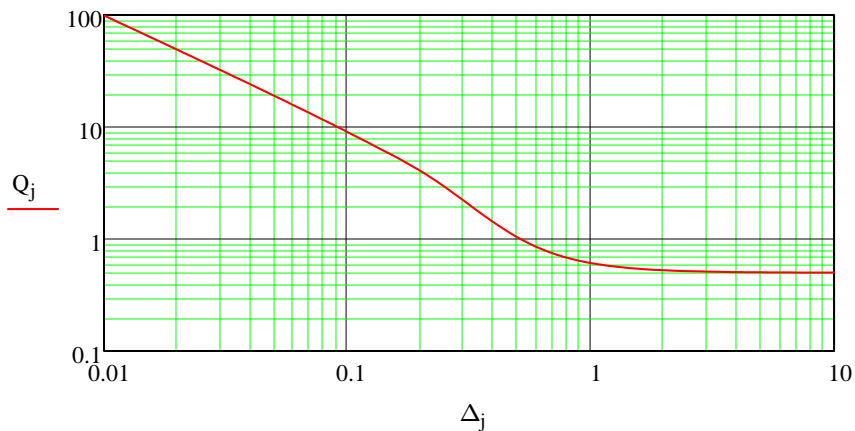
$$-k^2 \cdot \rho \cdot c^2 + i \cdot \omega \cdot [(b - i \cdot \omega) \cdot \rho] = 0$$

$$k^2 = \frac{\omega^2 + i \cdot \omega \cdot b(\omega)}{c^2}$$

$$k = k_0 \cdot \sqrt{1 + \frac{i \cdot b(\omega)}{\omega}} = k_0 \cdot \sqrt{1 + i \cdot \Delta^2 \cdot G(\Delta)} \quad k_0 = \frac{\omega}{c}$$

$$Q(\Delta) = \frac{1}{2} \cdot \frac{\operatorname{Re}(k)}{\operatorname{Im}(k)}$$

$$j := 0..100 \quad \Delta_j := .01 \cdot \left(\frac{10}{.01}\right)^{\frac{j}{100}} \quad rf_j := \sqrt{1 + i \cdot (\Delta_j)^2 \cdot G(\Delta_j)} \quad Q_j := \frac{1}{2} \cdot \frac{\operatorname{Re}(rf_j)}{\operatorname{Im}(rf_j)}$$



3) Finally, drive the motion by an equivalent mass influx uniformly distributed in the gap.

Rectangular Flow-Driven Gap

Gap oscillating as $g(t) = g \cdot (1 + \eta(x) \cdot e^{-i \cdot \omega \cdot t})$ $\eta(x) = \eta_0$ in the gap

Convert the oscillating gap to a fixed gap with an equivalent oscillatory gas source.

fluid acceleration $\frac{\delta v}{\delta t} = -\frac{1}{\rho} \cdot \frac{\delta P}{\delta x} - b(\omega) \cdot v$

conservation of mass $\frac{1}{c^2} \cdot \frac{\delta P}{\delta t} = -\rho \cdot \left(\frac{\delta v}{\delta x} + \frac{1}{g} \cdot \frac{\delta g}{\delta t} \right)$

The driven solution with the source = 0 and the pressure = 0 at the gap boundaries will be a Fourier series that is constant in the gap and opposite just outside the gap, periodically reversing for the Fourier series...

Square wave function, η_0 for $-\frac{L}{2} < x < \frac{L}{2}$

going to zero at the boundaries:

$$\eta(x) = 4 \cdot \eta_0 \cdot \sum_{(n=\text{odd})} \left[\frac{(-1)^{\frac{n-1}{2}}}{n \cdot \pi} \cdot \cos\left(\frac{n \cdot \pi \cdot x}{L}\right) \right]$$

sum over n odd $P = P_n \cdot \cos(q_n \cdot x) \cdot e^{-i \cdot \omega \cdot t}$ $v = v_n \cdot \sin(q_n \cdot x) \cdot e^{-i \cdot \omega \cdot t}$ $\frac{\delta g}{\delta t} = -i \cdot \omega \cdot g \cdot \eta(x) \cdot e^{-i \cdot \omega \cdot t}$

$$-i \cdot \omega \cdot v_n \cdot \sin(q_n \cdot x) = \frac{1}{\rho} \cdot q_n \cdot P_n \cdot q_n \cdot \sin(q_n \cdot x) - b(\omega) \cdot v_n \cdot \sin(q_n \cdot x)$$

$$\frac{-i \cdot \omega}{c^2} \cdot P_n \cdot \cos(q_n \cdot x) = -\rho \cdot \left[q_n \cdot v_n \cdot \cos(q_n \cdot x) + \frac{1}{g} \cdot (-i \cdot \omega \cdot g \cdot \eta_n \cdot \cos(q_n \cdot x)) \right]$$

$$-i \cdot \omega \cdot v = \frac{1}{\rho} \cdot q \cdot P - b(\omega) \cdot v$$

Temporarily drop the n index.

$$\frac{-i \cdot \omega}{c^2} \cdot P = -\rho \cdot \left[q \cdot v + \frac{1}{g} \cdot (-i \cdot \omega \cdot g \cdot \eta) \right]$$

$$\begin{pmatrix} \frac{q}{\rho} & i \cdot \omega - b \\ -i \cdot \frac{\omega}{\rho \cdot c^2} & q \end{pmatrix} \cdot \begin{pmatrix} P \\ v \end{pmatrix} = \begin{pmatrix} 0 \\ i \cdot \omega \cdot \eta \end{pmatrix}$$

$$\begin{pmatrix} P \\ v \end{pmatrix} = \frac{\begin{bmatrix} (\omega^2 + i \cdot \omega \cdot b) \cdot \rho \cdot c^2 \\ i \cdot \omega \cdot q \cdot c^2 \end{bmatrix}}{c^2 \cdot q^2 - \omega^2 - i \cdot \omega \cdot b} \cdot \eta$$

Check out the case of $b=0$ for getting our bearings...

$$\begin{pmatrix} P \\ v \end{pmatrix} = \frac{\begin{pmatrix} \omega^2 \cdot \rho \cdot c^2 \\ i \cdot \omega \cdot q \cdot c^2 \end{pmatrix}}{c^2 \cdot q^2 - \omega^2} \cdot \eta$$

$$\eta(x) = 4 \cdot \eta_0 \cdot \sum_{(n=\text{odd})} \left[\frac{(-1)^{\frac{n-1}{2}}}{n \cdot \pi} \cdot \cos\left(\frac{n \cdot \pi \cdot x}{L}\right) \right]$$

$$P(x) = 4 \cdot \eta_0 \cdot \omega^2 \cdot \rho \cdot c^2 \cdot \sum_{(n=\text{odd})} \left[\frac{(-1)^{\frac{n-1}{2}}}{n \cdot \pi \cdot \left[c^2 \cdot \left(\frac{n \cdot \pi}{L}\right)^2 - \omega^2 \right]} \cdot \cos\left(\frac{n \cdot \pi \cdot x}{L}\right) \right]$$

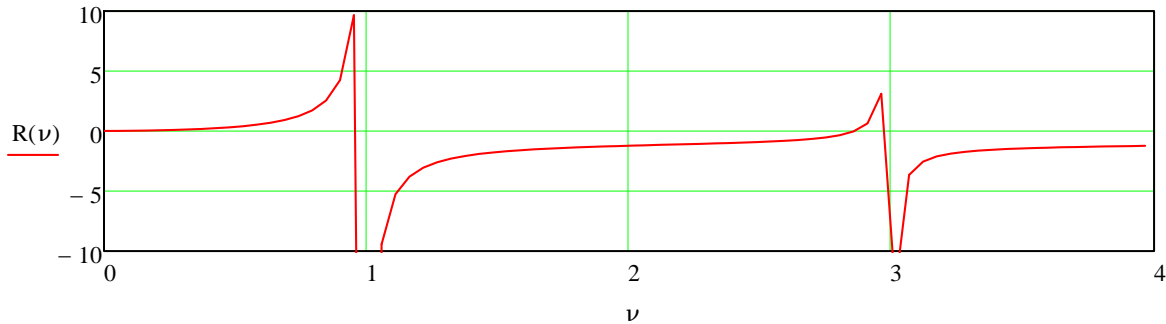
Force per length on the plate...

$$F = 2 \cdot \int_0^{\frac{L}{2}} P(x) dx = 8 \cdot L \cdot \eta_0 \cdot \omega^2 \cdot \rho \cdot c^2 \cdot \sum_{(n=\text{odd})} \left[\frac{1}{(n \cdot \pi)^2 \cdot \left[c^2 \cdot \left(\frac{n \cdot \pi}{L}\right)^2 - \omega^2 \right]} \right]$$

$$F(\omega) = \frac{8 \cdot L \cdot \eta_0 \cdot \rho \cdot c^2}{\pi^2} \cdot \nu^2 \cdot \sum_{(n=\text{odd})} \left[\frac{1}{n^2 \cdot (n^2 - \nu^2)} \right] \quad \nu = \frac{\omega \cdot L}{\pi \cdot c} \quad \omega = \frac{\pi \cdot c \cdot \nu}{L}$$

$$n := 1, 3 \dots 100 \quad R(\nu) := \nu^2 \cdot \sum_n \left[\frac{1}{n^2 \cdot (n^2 - \nu^2)} \right]$$

$$\nu := 0, .052875 \dots 4$$



At low frequencies, the gas flows into the gap for positive η . As the first resonance is approached, the amplitude of the gas in-flow increases, resulting in the positive pressure-to- η phase relation. After the resonance, the phase is reversed. This negative phase relation is what would be expected for trapped gas undergoing rarefaction (negative pressure) with increasing ν .

Now include the damping term b ...

$$P_n = \frac{(\omega^2 + i \cdot \omega \cdot b) \cdot \rho \cdot c^2}{c^2 \cdot q^2 - \omega^2 - i \cdot \omega \cdot b} \cdot \eta_n$$

$$P(x) = 4 \cdot \eta_0 \cdot (\omega^2 + i \cdot \omega \cdot b)^2 \cdot \rho \cdot c^2 \cdot \sum_{(n=\text{odd})} \left[\frac{(-1)^{\frac{n-1}{2}}}{n \cdot \pi \cdot \left[c^2 \cdot \left(\frac{n \cdot \pi}{L} \right)^2 - \omega^2 - i \cdot \omega \cdot b \right]} \cdot \cos\left(\frac{n \cdot \pi \cdot x}{L}\right) \right]$$

Force per length on the plate...

$$F = 2 \cdot \int_0^{\frac{L}{2}} P(x) dx = 8 \cdot L \cdot \eta_0 \cdot (\omega^2 + i \cdot \omega \cdot b)^2 \cdot \rho \cdot c^2 \cdot \sum_{(n=\text{odd})} \left[\frac{1}{(n \cdot \pi)^2 \cdot \left[c^2 \cdot \left(\frac{n \cdot \pi}{L} \right)^2 - \omega^2 - i \cdot \omega \cdot b \right]} \right]$$

$$\frac{b}{\omega} = \frac{2 \cdot \gamma \cdot L}{\pi \cdot c \cdot \nu \cdot g^2} \cdot G(\Delta) = \Delta^2 \cdot G(\Delta) \quad 1 + i \cdot \Delta^2 \cdot G(\Delta) = \frac{1}{1 - \frac{2 \cdot \Delta}{1 - i} \cdot \tanh\left(\frac{1 - i}{2 \cdot \Delta}\right)}$$

$$\nu^2 \cdot \left(1 + i \cdot \frac{b}{\omega}\right) = \frac{\nu^2}{1 - \frac{2}{1 - i} \cdot \sqrt{\frac{\nu_0}{\nu}} \cdot \tanh\left(\frac{1 - i}{2} \cdot \sqrt{\frac{\nu}{\nu_0}}\right)} \quad \Delta = \sqrt{\frac{\nu_0}{\nu}} \quad \nu_0 = \frac{2 \cdot \gamma \cdot L}{\pi \cdot c \cdot \rho \cdot g^2}$$

$$F(\omega) = \frac{8 \cdot L \cdot \eta_0 \cdot \rho \cdot c^2}{\pi^2} \cdot \frac{\nu^2}{1 - \frac{2}{1 - i} \cdot \sqrt{\frac{\nu_0}{\nu}} \cdot \tanh\left(\frac{1 - i}{2} \cdot \sqrt{\frac{\nu}{\nu_0}}\right)} \cdot \sum_{(n=\text{odd})} \left[\frac{1}{n^2 \cdot \left(n^2 - \frac{\nu^2}{1 - \frac{2}{1 - i} \cdot \sqrt{\frac{\nu_0}{\nu}} \cdot \tanh\left(\frac{1 - i}{2} \cdot \sqrt{\frac{\nu}{\nu_0}}\right)} \right)} \right]$$

Now put in some real numbers... $p = \frac{P}{1 \cdot \text{ATm}} \quad p := 0.25$

Fits to NIST Thermodata

$$c(p) := 179.342 \cdot \left(1 - \frac{p}{40.3}\right) \quad \gamma(p) := 5.37 \cdot 10^{-6} \cdot \left(1 + \frac{p}{84}\right) \quad \rho(p) := 4.405 \cdot p \cdot \left(1 + \frac{p}{21.5}\right)$$

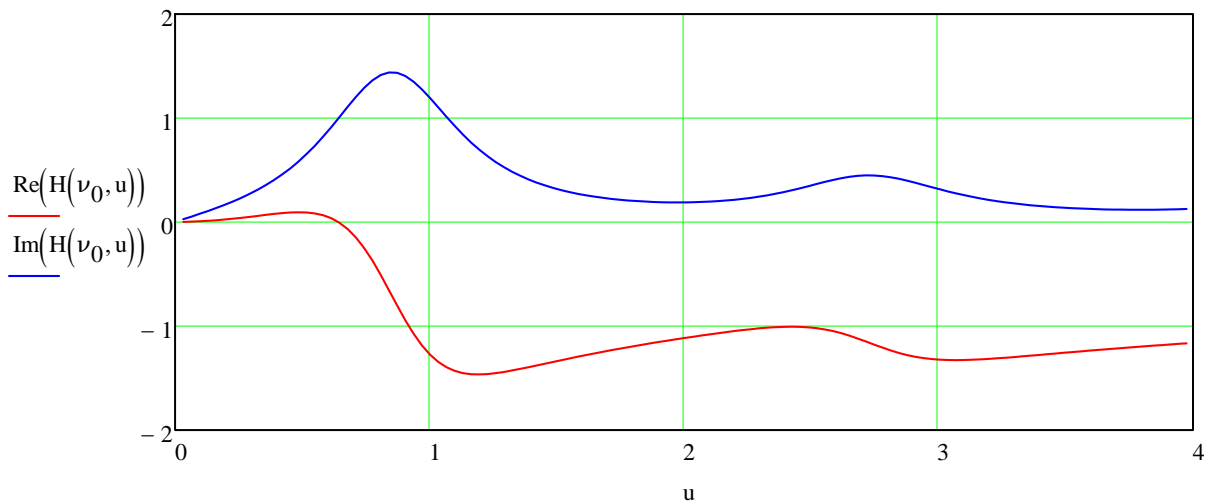
$$L := 0.003 \quad w := 0.02 \quad g := 20 \cdot 10^{-6} \quad \omega := 2 \cdot \pi \cdot 34000 \quad \nu := \frac{\omega \cdot L}{\pi \cdot c(p)} = 1.145$$

$$\nu_0 := \frac{2 \cdot \gamma(p) \cdot L}{\pi \cdot c(p) \cdot \rho(p) \cdot g^2} \quad \nu_0 = 0.13 \quad M_p := \pi \cdot (0.0065^2 - 0.004^2) \cdot 0.5 \cdot 3.$$

$$F = \frac{8 \cdot L \cdot w \cdot \eta_0 \cdot \rho \cdot c^2}{\pi^2} \cdot H(\nu)$$

$$H(\nu_0, \nu) := \left| \begin{array}{l} u \leftarrow \frac{\nu^2}{1 - \frac{2}{1-i} \cdot \frac{\sqrt{\nu_0}}{\sqrt{\nu}} \cdot \tanh\left(\frac{1-i}{2} \cdot \frac{\sqrt{\nu}}{\sqrt{\nu_0}}\right)} \\ \Sigma \leftarrow 0 \\ \text{for } n \in 1, 3 \dots 83 \\ \Sigma \leftarrow \Sigma + \frac{1}{n^2 \cdot (n^2 - u)} \\ u \cdot \Sigma \end{array} \right.$$

$$u := 0, .03231 \dots 4$$



This is the force with viscous loss, behaving the same as the $b=0$ case, but now with the imaginary part.

The real part of the scaled force is opposite to the displacement for our situation (around $\nu=2$). This is an extra restoring force due to the gas compression. The average loss comes from the positive imaginary part of the force:

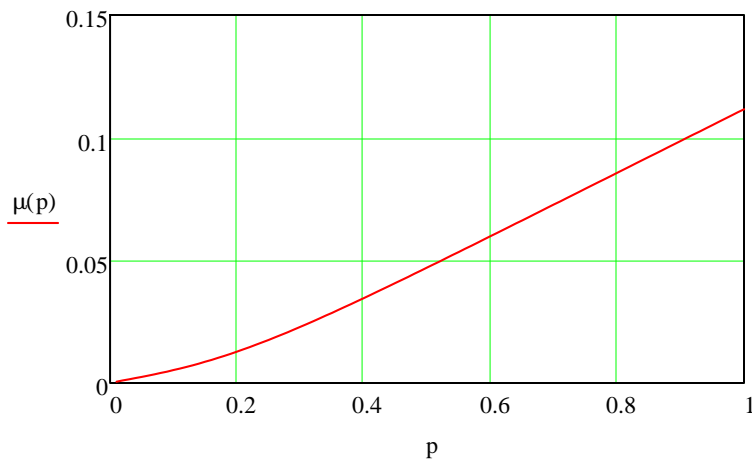
$$p := 0.01, 0.02 \dots 1$$

Effective Mass $f = -M \cdot \omega^2 \cdot x$

$$\frac{8 \cdot L \cdot w \cdot \eta_0 \cdot \rho \cdot c^2}{\pi^2} \cdot \operatorname{Re}(H(\nu_0, \nu)) = -M \cdot \omega^2 \cdot g \cdot \eta$$

$$M = -\frac{8 \cdot L^3 \cdot w \cdot \rho}{\pi^4 \cdot g} \cdot \frac{\operatorname{Re}(H(\nu_0, \nu))}{\nu^2}$$

$$\mu(p) := -\frac{8 \cdot L^3 \cdot w \cdot \rho(p)}{\pi^4 \cdot g \cdot M_p} \left| \begin{array}{l} \nu_0 \leftarrow \frac{2 \cdot \gamma(p) \cdot L}{\pi \cdot c(p) \cdot \rho(p) \cdot g^2} \\ \nu \leftarrow \frac{\omega \cdot L}{\pi \cdot c(p)} \\ \frac{\operatorname{Re}(H(\nu_0, \nu))}{\nu^2} \end{array} \right.$$



Damping

$$F = \frac{8 \cdot L \cdot w \cdot \eta_0 \cdot \rho \cdot c^2}{\pi^2} \cdot H(\nu)$$

$$W = -\frac{1}{2} \cdot \operatorname{Re}(F \cdot \bar{\nu}) = -\frac{1}{2} \cdot \operatorname{Re}\left[F \cdot \overline{(-i \cdot \omega \cdot \eta_0 \cdot g)}\right] = \frac{1}{2} \cdot \omega \cdot \eta_0 \cdot g \cdot \operatorname{Im}(F)$$

$$E = 2 \cdot \frac{1}{2} \cdot M_p \cdot (\omega \cdot g \cdot \eta_0)^2$$

$$Q = \omega \cdot \frac{E}{W} = \omega \cdot \frac{2 \cdot \frac{1}{2} \cdot M_p \cdot (\omega \cdot g \cdot \eta_0)^2}{\frac{1}{2} \cdot \omega \cdot \eta_0 \cdot g \cdot \operatorname{Im} \left(\frac{8 \cdot L \cdot w \cdot \eta_0 \cdot \rho \cdot c^2}{\pi^2} \cdot H(\nu) \right)} = \frac{\pi^4 \cdot g \cdot M_p}{4 \cdot w \cdot \rho \cdot L^3} \cdot \frac{\nu^2}{\operatorname{Im}(H(\nu))}$$

$$Q(p) := \frac{\pi^4 \cdot g \cdot M_p}{4 \cdot w \cdot \rho(p) \cdot L^3} \cdot \left| \begin{array}{l} \nu_0 \leftarrow \frac{2 \cdot \gamma(p) \cdot L}{\pi \cdot c(p) \cdot \rho(p) \cdot g^2} \\ \nu \leftarrow \frac{\omega \cdot L}{\pi \cdot c(p)} \\ \frac{\nu^2}{\operatorname{Im}(H(\nu_0, \nu))} \end{array} \right.$$

